


# Flow Control: TD4-1

## *Overview*

- General issues, passive vs active...
- Control issues: optimality and learning vs robustness and rough model
- Model-based control: linear model, nonlinear control
  - Linear model, identification
  - Sliding Mode Control
  - Delay effect
  - Time-delay systems
- Introduction to delay systems
  - Examples
  - Much a do about delay? Some special features + a bit of maths
  - Time-varying delay
-  - Model-based control: nonlinear model, nonlinear control
  - Overview of MF's PhD: Sliding Mode Control
  - Application to the airfoil
  - Application to the Ahmed body (MF and CC'PhDs)
- ...
- Machine Learning and model-free control: + 4h with [Thomas Gomez](#)

# Flow Control

Nonlinear active control of turbulent separated flows: theory and experiments.

Maxime FEINGESICHT



<https://tel.archives-ouvertes.fr/tel-01801155>

Lille, France - December 11, 2017

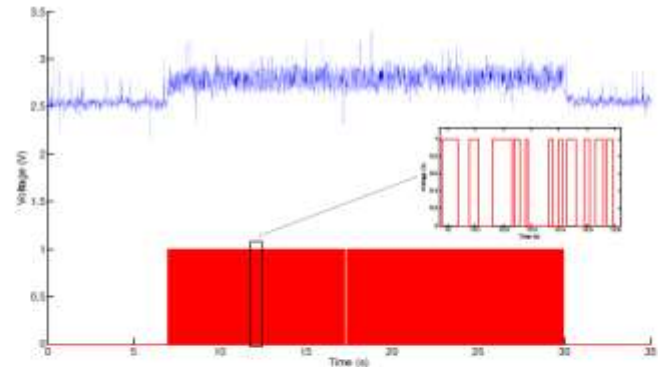
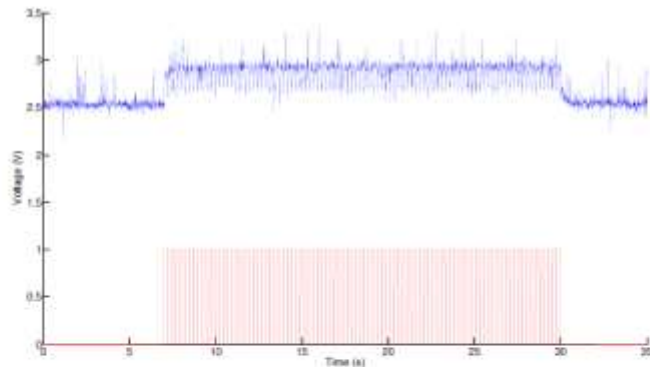
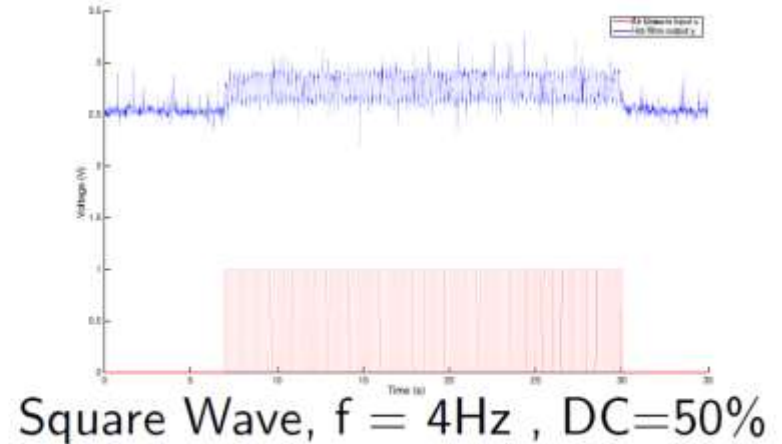
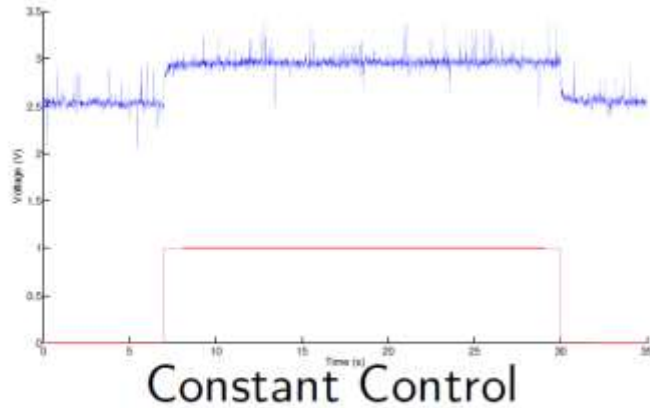
Supervisor: Jean-Pierre RICHARD

Co-supervisors: Andrey POLYAKOV, Franck KERHERVE



Back to the results from Maxime's PhD:  
nonlinear delayed model with Sliding Mode Control

# 1) Model: identification



Square Wave,  $f = 4\text{Hz}$ ,  $\text{DC}=80\%$   
 $y(t) = y_c(t) - y_0(t)$ , with  $y_c(t)$  the current hot-film output and  $y_0(t)$  the hot-film output with no control (approximately 2.5V).

# 1) Model: identification

Back to  
slide 36



## Least-square identification [Ljung 1999] for fixed delays

$$\left\{ \begin{array}{l}
 W = \begin{bmatrix}
 y_0 - \tau_1 & \dots & y_0 - \tau_{N_1} & u_0 - h_1 & \dots & u_0 - h_{N_3} & y_0 - \tau_1 u_0 - h_1 & y_0 - \tau_2 u_0 - h_1 & \dots & y_0 - \tau_{N_2} u_0 - h_{N_3} \\
 \vdots & & \vdots & \vdots & & \vdots & \vdots & \vdots & & \vdots \\
 y_{(N-1) - \tau_1} & \dots & y_{(N-1) - \tau_{N_1}} & u_{(N-1) - h_1} & \dots & u_{(N-1) - h_{N_3}} & y_{(N-1) - \tau_1} u_{(N-1) - h_1} & y_{(N-1) - \tau_2} u_{(N-1) - h_1} & \dots & y_{(N-1) - \tau_{N_2}} u_{(N-1) - h_{N_3}}
 \end{bmatrix} \\
 A = [a_0 \quad \dots \quad a_{N_1} \quad b \quad c_0 \quad \dots \quad c_{N_2}]^T \\
 x = [y_1 \quad \dots \quad y_N]^T \\
 WA = x \Rightarrow A = W^+ x ; W^+ = \text{Moore-Penrose inverse of } W
 \end{array} \right. \quad (3)$$

$N \sim 10^5$   
to  $3 \cdot 10^6$

Issue :  $\min_{\tau, \bar{\tau}, h, A} \|y^{simu} - WA\|_2$  or  $\max_{\tau, \bar{\tau}, h, A} \text{FIT}(y) \Rightarrow$  Necessity to use a nonlinear mixed integer programming algorithm, e.g. NOMAD ([Le Digabel 2011]) or Genetic Algorithm. The FIT coefficient is defined as (see [Dandois 2013]) :

$$\text{FIT}(y) = 1 - \sqrt{\frac{\sum_{k=1}^N (y_k - y_k^{simu})^2}{\sum_{k=1}^N (y_k - \bar{y}_k)^2}}$$
 , with  $\bar{y}_k$  the average value of  $y$  up to the time step  $k$  and  $y_k^{simu}$  the output value of the model.

# 1) Model: identification

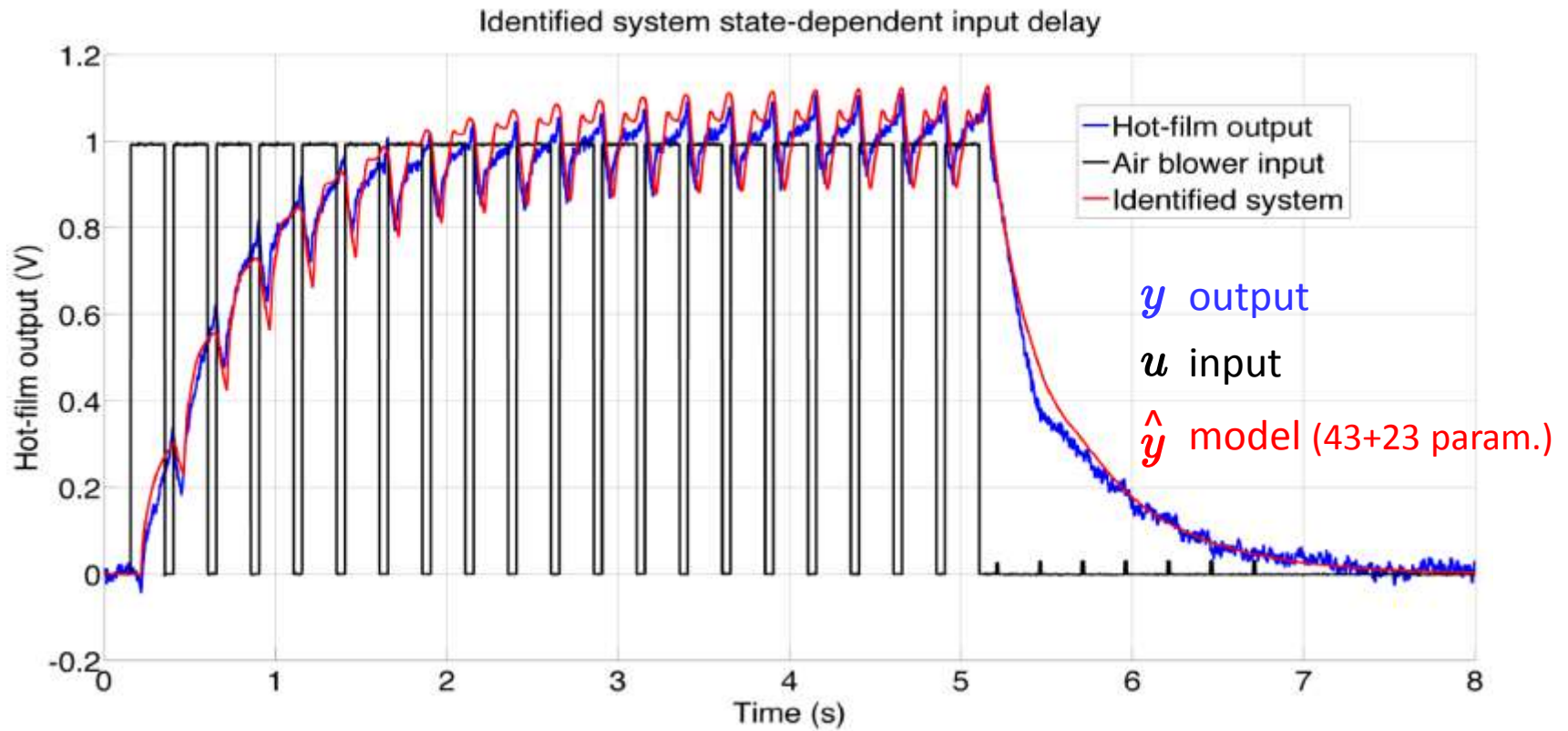
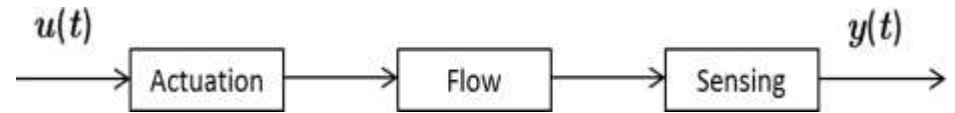
For a model with a FIT of approximately 80%, the coefficient vectors have the following sizes:

- $\tau$  and  $a$  : vectors of size 10
- $\tilde{\tau}$  and  $c$  : vectors of size 10
- $h$  and  $b$  : vectors of size 3
- Total number of coefficients to find : 43 coefficients and 23 delays

*Fluid Mechanics literature* (with similar FIT) → 1000 to 10000 coefficients

NB: for our robust controller, 4 coefs + 3 delays = **7 !**

# 1) Model: identification



## 2) Control: a simplified model

4 coefs + 3 delays = 7 parameters

### Continuous delayed bilinear model

$$\begin{aligned} \dot{y}(t) = & a_1 y(t-h) - a_2 y(t-\tau) \\ & + (b - cy(t-h) + cy(t-\bar{\tau})) u(t-h) \end{aligned} \quad (9)$$

Proposition 2: Positivity and boundedness of the system  
(Feingesicht et al. 2017b; Feingesicht et al. 2017c)

If  $c < a_1$ ,  $(a_1 + c)\tau < a_2\tau < \frac{1}{e}$  and  $\tau \leq h \leq \bar{\tau}$   
then the system (9) is positive and its solution is globally bounded  
for any input signal  $u \in \mathbf{L}^\infty : u(t) \in \{0, 1\}$  as follows

$$0 \leq y(t) \leq y_{max} = \frac{b}{a_2 - a_1} \text{ for all } t \geq 0$$

## 2) Control: surface for the Sliding Mode Control

### Sliding variable

We choose the following sliding variable:

$$\sigma(t) = y(t) - a_2 \int_{t-\tau}^t y(s) ds + c \int_{t-\bar{\tau}+h}^t y(s) ds$$

Does that sound familiar?

Yes, predictor effect ☺

(slide 63)

$$+ \int_{t-h}^t a_1 y(s) + (b - cy(s) + cy(s-\bar{\tau}+h))u(s) ds$$

It satisfies the equation:

$$\dot{\sigma}(t) = (a_1 - a_2 + c(1 - u(t)))y(t) + c(u(t) - 1)y(t - \bar{\tau} + h) + bu(t)$$

There is no delay in the control input  $u$  in the expression of the derivative of the sliding variable  $\dot{\sigma}$ . We use the sliding mode existence condition  $\dot{\sigma}(t)\sigma(t) < 0$  from [Utkin, 1992] for control design.



## 2) Control: conditions for Sliding Mode

Proposition 3: Conditions for Sliding Mode (Feingessicht et al. 2017b; Feingessicht et al. 2017c)

If conditions of Proposition 2 hold and

$$Q(j\omega) \neq 0 \quad \text{for } \omega \neq 0,$$

where  $Q(s) = s + a_2 e^{-s\tau} - (a_2 - c)e^{-sh} - ce^{-s\bar{\tau}}$ ,  $s \in \mathbb{C}$  and  $j^2 = -1$ ,  
then the control law

$$u(t) = \begin{cases} 1 & \text{if } \sigma(t) < \sigma^*, \\ 0 & \text{if } \sigma(t) > \sigma^*, \end{cases}$$

with  $\sigma^* = y^*(1 + a_2(h - \tau) + c(\bar{\tau} - h))$  and  $y^* \in \left(0, \frac{b}{a_2 - a_1}\right)$

guarantees  $y(t) \rightarrow y^*$  as  $t \rightarrow +\infty$ .

A more general version of this proposition for any number of delays is presented in the patent: M. Feingessicht, A. Polyakov, F.

Kerherve and J. P. Richard, **Contrôle par modes glissants du décollement d'un écoulement**, FR1755440, deposited

15/06/2017

## 2) Control: proof of robustness w.r.t. additive perturbation

### Corollary 1: Robustness

Let the model (9) contain an additive perturbation:

$$\begin{aligned} \dot{y}(t) = & a_1 y(t-h) - a_2 y(t-\tau) \\ & + (b - cy(t-h) + cy(t-\bar{\tau})) u(t-h) + d(t) \end{aligned} \quad (10)$$

where  $d(t)$  is a bounded signal  $\|d(t)\| < \bar{d}$  such that (10) remains a positive system. If

$$\bar{d} < \frac{b}{1 + \frac{a_2 - a_1}{a_2 - a_1 - c}}$$

then  $y(t) \rightarrow y^* + O(h\bar{d})$  as  $t \rightarrow +\infty$  for  $y^* \in \left( \frac{\bar{d}}{a_2 - a_1 - c}, \frac{b - \bar{d}}{a_2 - a_1} \right)$ .

### Conjecture for future developments

The condition

$$Q(j\omega) \neq 0 \quad \text{for } \omega \neq 0,$$

may not be necessary and can be removed from Proposition 3.

## 2) Control: summary

$$\sigma(t) = y(t) - a_2 \int_{t-\tau}^t y(s) ds + c \int_{t-\bar{\tau}+h}^t y(s) ds + \int_{t-h}^t a_1 y(s) + (b - cy(s) + cy(s-\bar{\tau}+h)) u(s) ds$$

### Proposition 2 : Conditions for Sliding Mode

If conditions of Proposition 1 hold and

$$Q(j\omega) \neq 0 \quad \text{for} \quad \omega \neq 0, \quad (7)$$

where  $Q(s) = s + a_2 e^{-s\tau} - (a_2 - c)e^{-sh} - ce^{-s\bar{\tau}}$ ,  $s \in \mathbb{C}$  and  $j^2 = -1$ , then the control law

$$u(t) = \begin{cases} 1 & \text{if } \sigma(t) < \sigma^*, \\ 0 & \text{if } \sigma(t) > \sigma^*, \end{cases} \quad (8)$$

with  $\sigma^* = y^*(1 + a_2(h - \tau) + c(\bar{\tau} - h))$  and  $y^* \in \left(0, \frac{b}{a_2 - a_1}\right)$  guarantees

$y(t) \rightarrow y^*$  as  $t \rightarrow +\infty$ .

## 2) Control: check on simulation

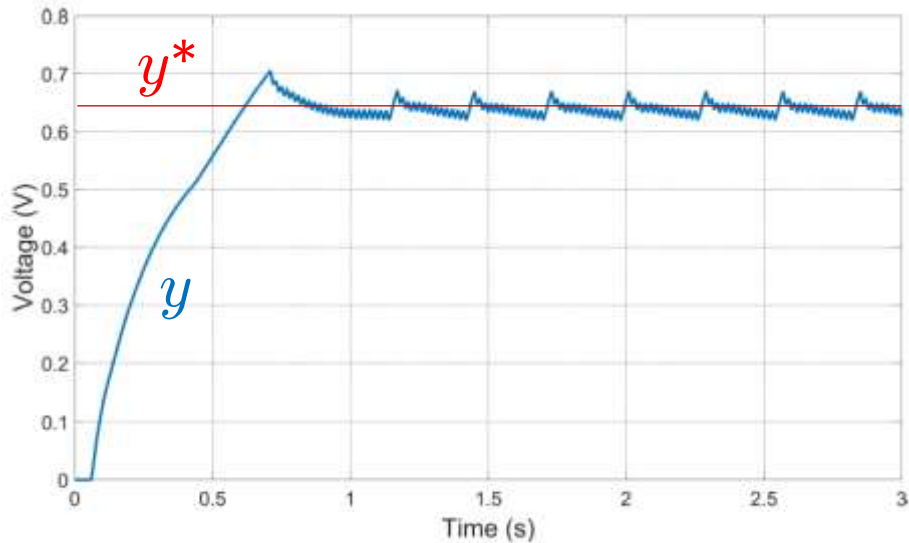


Figure 6. Application of the setpoint tracking control: Output of the system

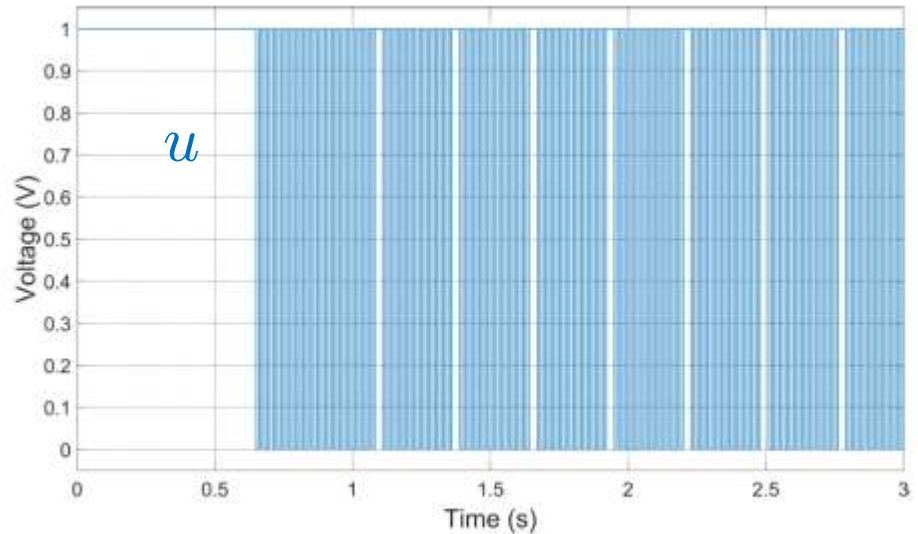


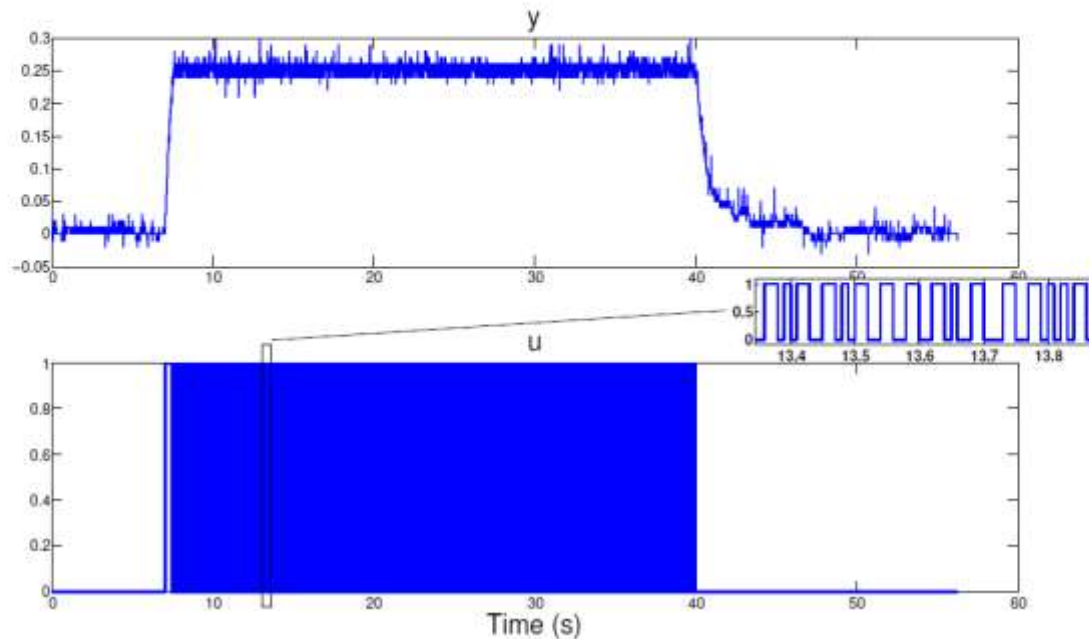
Figure 7. Application of the setpoint tracking control: Control signal

Jan. 2015: PhD starts  
March 2016: control ok on simulations ☺  
October: journal paper submitted ☺  
December: still looking for platform's availability... ☹  
Feb. 2017: actuated platform ready!

# 3) Experiments!

$$v = 34.5m/s ; AoA = 24^\circ ; q = 25g/s ; y^* = 0.25$$

The model was identified for a flow speed  $v = 34.5m/s$ , an angle of attack  $AoA = 24^\circ$  and an actuator mass flow  $q = 25g/s$  (nominal conditions).



nominal

# 3) Experiments

<https://www.youtube.com/watch?v=b5NnAV2qeno>

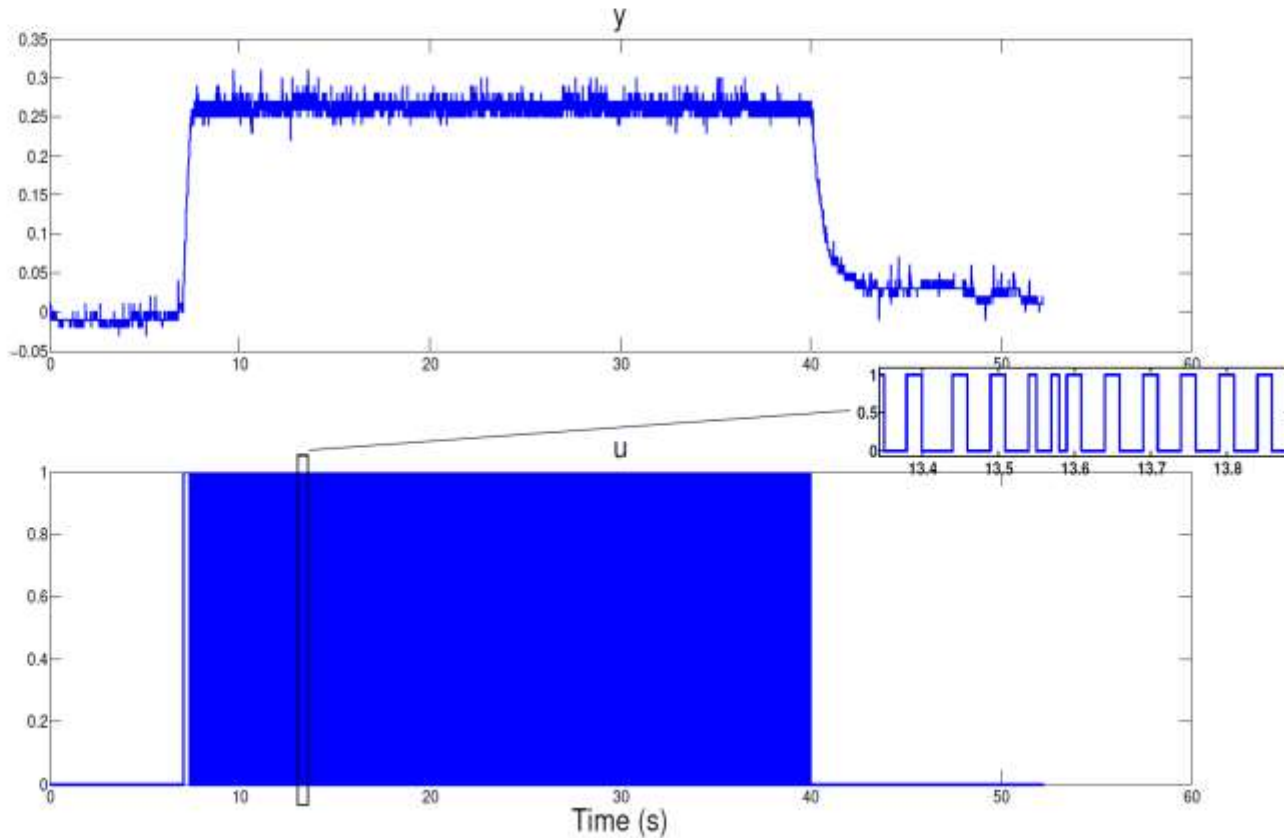
WITHOUT CONTROL:  
TURBULENT FLOW

---

$v = 14.5 \text{ m/s}$  ;  $AoA = 24^\circ$  ;  $q = 25 \text{ g/s}$  ;  $y^* = 0.25$

v	flow speed
AoA	angle of the flap
q	mass flow of the actuators

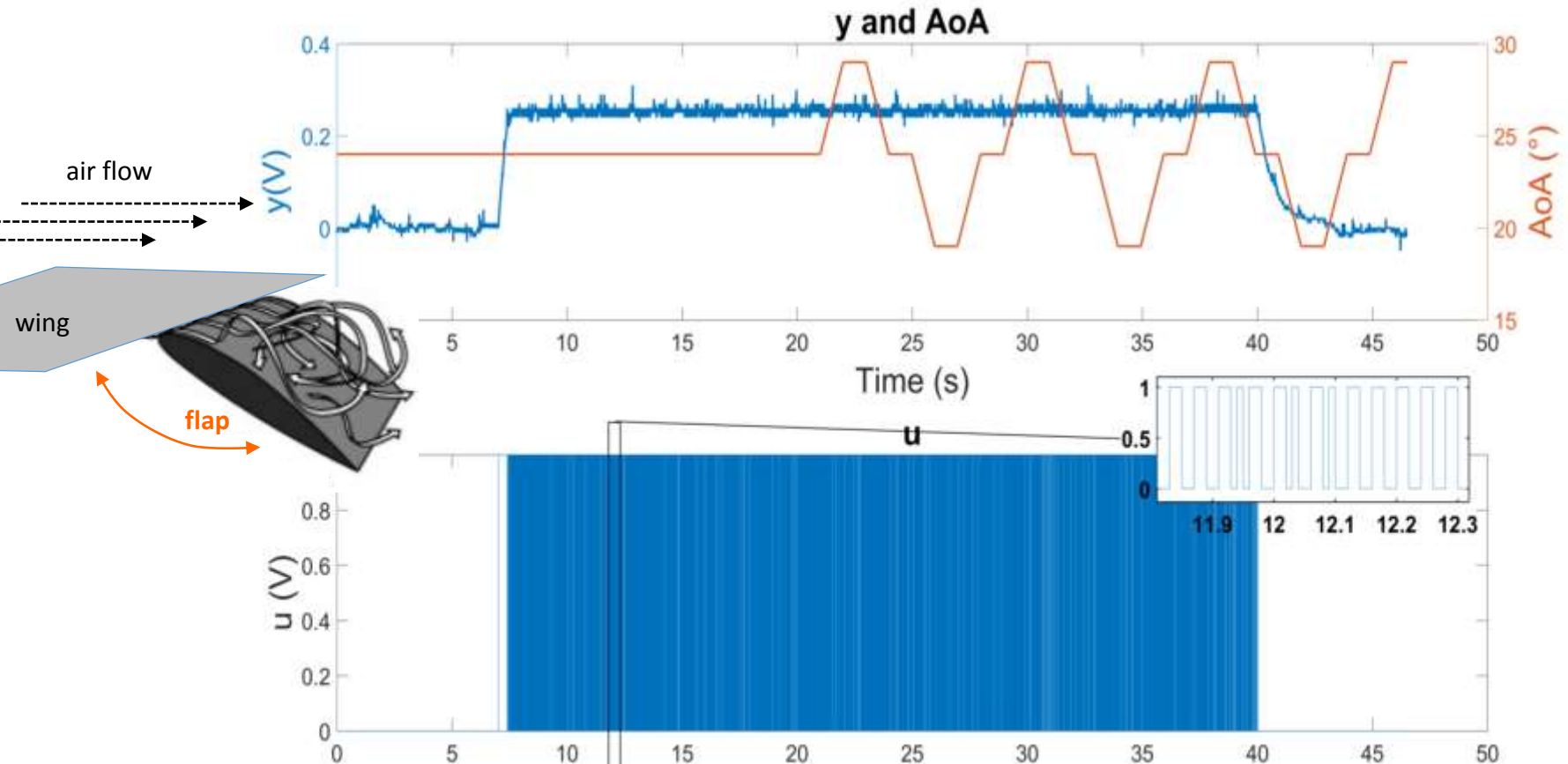
Non nominal:  
lower flow speed



$v = 34.5m/s$  ;  $AoA = \text{Time varying}$  ;  $q = 25g/s$  ;  
 $y^* = 0.25$

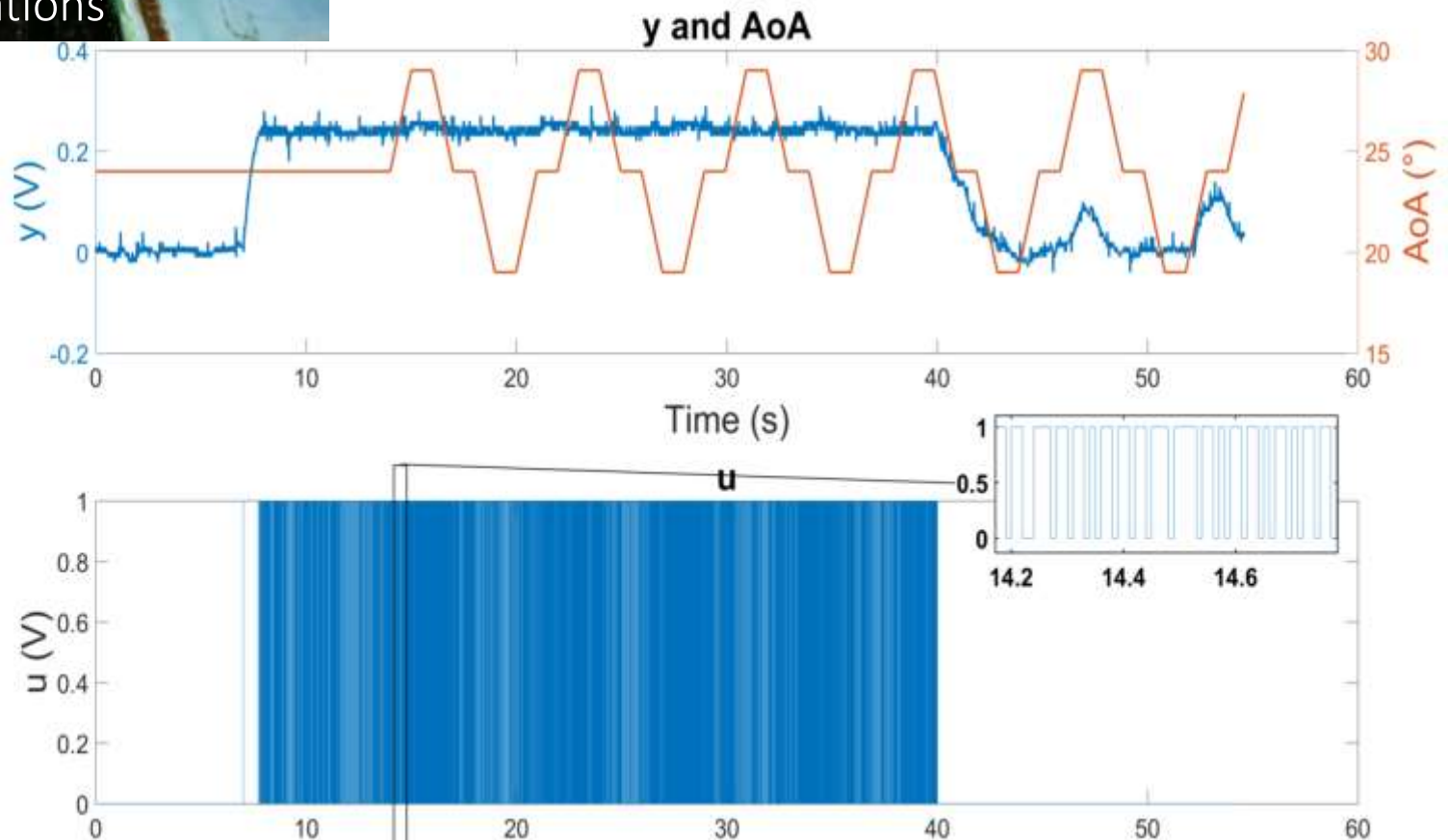
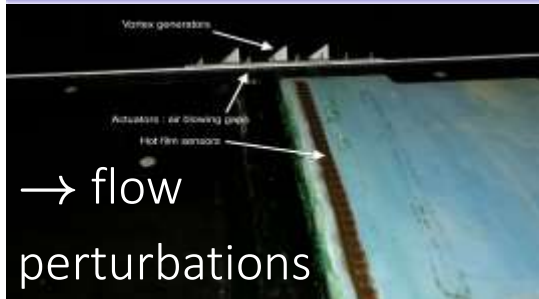
$v$	flow speed
$AoA$	angle of the flap
$q$	mass flow of the actuators

flap angle variations  
 $AoA = 20^\circ \text{ to } 38^\circ$





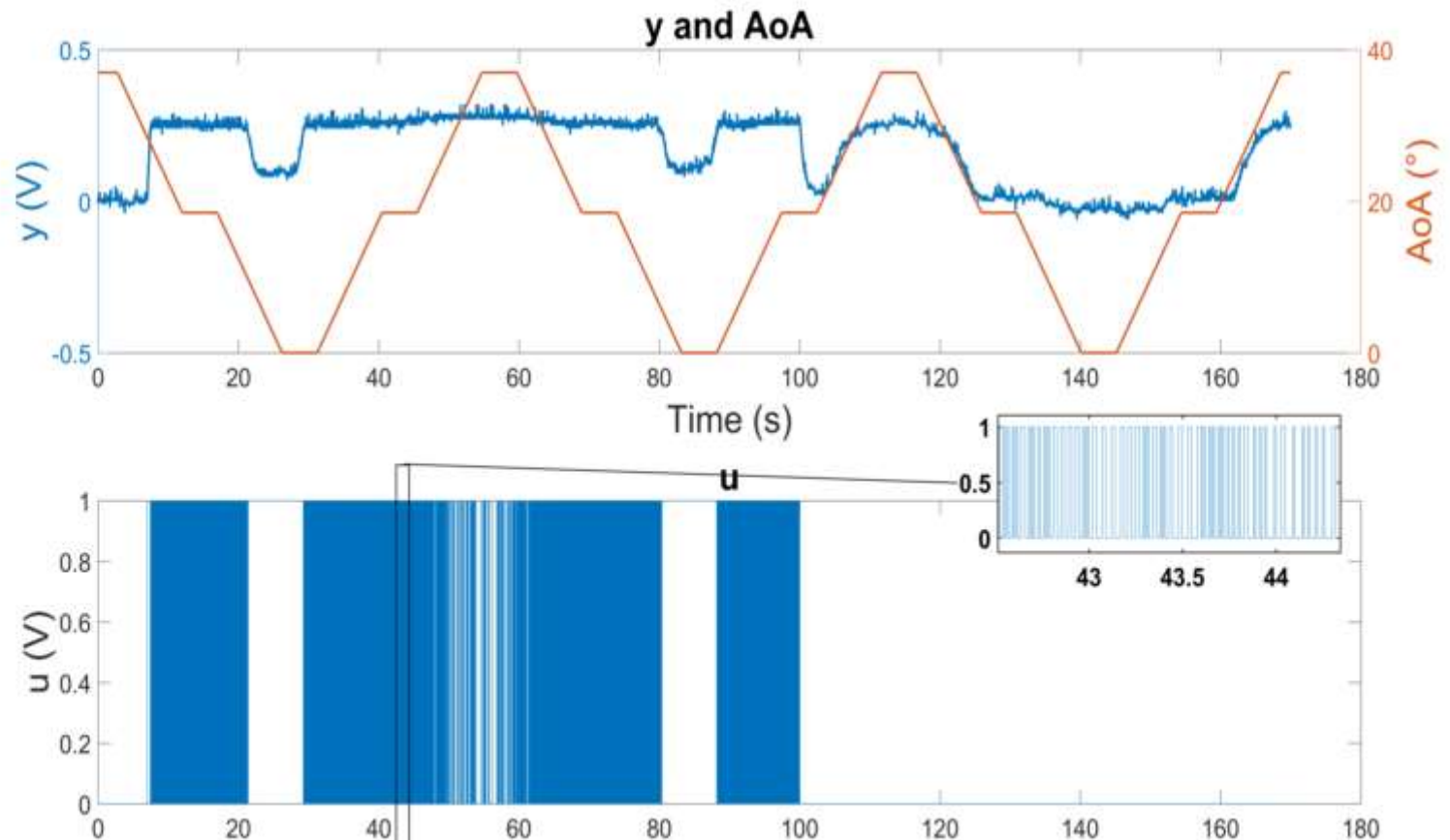
$v = 34.5\text{m/s}$  ;  $AoA = \text{Time varying}$  ;  $q = 25\text{g/s}$  ;  
 $y^* = 0.25$  ; vortex generators added before the flap



$v = 34.5\text{m/s}$  ;  $\text{AoA} = \text{Time varying}$  ;  $q = 25\text{g/s}$  ;  
 $y^* = 0.25$

$v$  flow speed  
 $\text{AoA}$  angle of the flap  
 $q$  mass flow of the actuators

largest angle variations  
 $\text{AoA} = 0^\circ$  to  $38^\circ$



# Flow Control: conclusions

- Simple models were developed for the studied flows
- These models come from intuition and have been shown to be an approximation of the Burgers equation
- The models can reach the same accuracy as the ones in the literature with 100 to 1000 times less coefficients
- Sliding Mode Control is perfectly adapted to the system and gives very good results with experimentally tested robustness
- The control was implemented using an Arduino, therefore it is not computationally or memory expensive

# Flow Control: perspectives

## Theoretical perspectives:

- Extension of the model and control to MIMO models
- Study the effect of discretization on the Sliding Mode control
- Use frequency analysis on the control signals and compare it to natural frequencies of the flow

## Practical perspectives:

- Application of the control in the MIMO case to the Ahmed body in LAMIH with measurement of drag and lift and 3 actuators
- Application on a real car with LAMIH-UVHC. The car is equipped with pressures sensors to estimate the drag and air jets actuators and is tested on French highway
- Application on flexible airplane (Michigan University), trains (Faiveley Transport) and drones (InterReg European project submitted with U. Southampton, TU Delft, UVHC, ONERA, Inria, Alstom, i-Trans, Railenium)